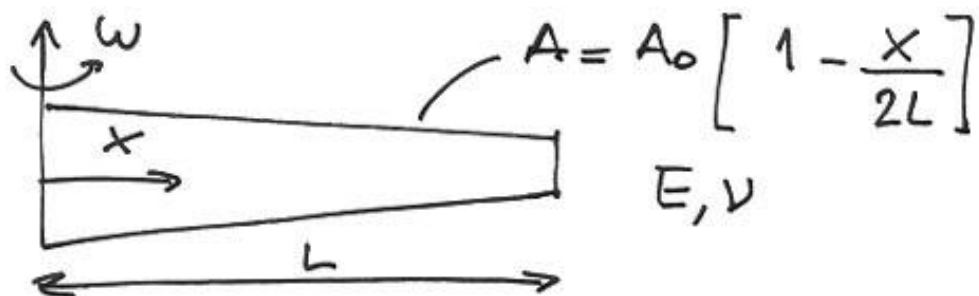


16.21 Final 2002 - Practice Session



Determination of the load on blade

$$a(x) = \omega^2 x$$

$$f(x) = \rho A(x) a(x) = \rho \omega^2 A_0 x \left(1 - \frac{x}{2L} \right)$$

$$\frac{\text{kg}}{\text{m}^3} \frac{1}{\text{s}^2} \text{m}^2 \text{m} = \frac{\text{N}}{\text{m}} \quad \checkmark$$

Continuum model

• differential: uniaxial deformations

$$(EA u')' + f(x) = 0$$

$$u(0) = 0 \quad \text{essential B.C.}$$

$$\underbrace{E u'(L) A(L)}_{\underbrace{\sigma(L)}_{N(L)}} = 0 \Rightarrow u'(L) = 0 \quad \text{natural B.C.}$$

(2)

• energy

$$U = \int_0^L \frac{1}{2} E u'^2 dx$$

$$V = - \int_0^L f(x) u(x) dx$$

$$\pi = U + V$$

$\delta\pi = 0 \Rightarrow$ equilibrium $\forall \delta u$ s.t. $\delta u = 0$ on $\partial\Omega_u$

$$\delta\pi = \int_0^L \frac{1}{2} EA 2 u' \delta u' dx - \int_0^L f(x) \delta u dx$$

$$= \int_0^L (EA u' \delta u)' dx - \int_0^L (EA u')' \delta u dx - \delta V$$

$$0 = EA u' \delta u \Big|_0^L - \int_0^L (EA u')' \delta u dx - \int_0^L f(x) \delta u dx$$

$\Rightarrow (EA u')|_L = 0$ natural B.C.

$$(EA u')' + f(x) = 0 \quad 0 < x < L$$

(3)

• PVD (alternatively)

$$\int_0^L A \sigma \delta \epsilon = \int_0^L f(x) \delta u dx$$

• Approximate solution:

• Ritz method: $\phi_i = x^i$

$$u \approx C_i \phi_i$$

$$\Pi = \int_0^L \frac{E A_0}{2} \left(1 - \frac{x}{2L}\right) (\phi_i' C_i)^2 dx - \int_0^L \rho \omega^2 A_0 \left(1 - \frac{x}{2L}\right) \phi_i C_i dx$$

$$\frac{\partial \Pi}{\partial C_i} = 0$$

• One term: $u \approx C_1 x \quad u' = C_1$

$$\Pi = \int_0^L \frac{E A_0}{2} \left(1 - \frac{x}{2L}\right) C_1^2 dx - \int_0^L \rho \omega^2 A_0 \left(1 - \frac{x}{2L}\right) C_1 x dx$$

$$\frac{\partial \Pi}{\partial C_1} = C_1 \int_0^L E A_0 \left(1 - \frac{x}{2L}\right) dx - \int_0^L \rho \omega^2 A_0 \left(1 - \frac{x}{2L}\right) x dx$$

$$= C_1 E A_0 \left(L - \frac{L}{4}\right) - \rho \omega^2 A_0 \left(\frac{L^3}{3} - \frac{L^3}{8}\right)$$

$$= C_1 \frac{3}{4} E - \frac{5 \rho \omega^2 L^2}{24}$$

$$\Rightarrow C_1 = \frac{4 \cdot 5 \rho \omega^2 L^2}{3 \cdot 24 E} = \frac{5}{18} \frac{\rho \omega^2 L^2}{E}$$

Post processing:

$$u = C_1 \phi_1 = \frac{5}{18} \frac{\rho \omega^2 L^2}{E} x$$

$$\epsilon = u' = \frac{5}{18} \frac{\rho \omega^2 L^2}{E}$$

$$\sigma = \frac{5}{18} \rho \omega^2 L^2$$

• $L = 5$, $\rho = 2700$, $\sigma_y = 500 \text{ MPa}$

$$\sigma = \frac{5}{18} \times 2700 \times 25 \times \omega^2 \leq 500 \cdot 10^6$$

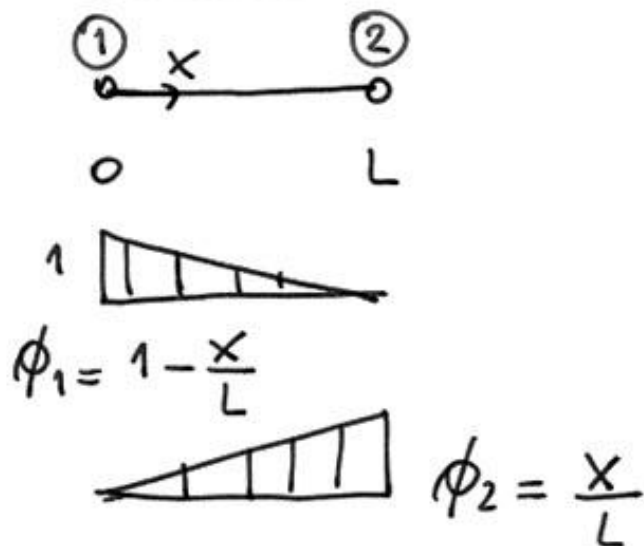
$$\omega^2 \leq 26666$$

$$\omega_{max} = 163 \frac{1}{\text{sec}}$$



(5)

• Finite elements



$$K = \int_0^L E A_0 \left(1 - \frac{x}{2L}\right) \begin{bmatrix} (-1/L)^2 & (-1/L)(1/L) \\ (-1/L)(1/L) & (1/L)^2 \end{bmatrix} dx$$

$$= \frac{E A_0}{L^2} \left(L - \frac{L^2}{4}\right) \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} = \frac{3}{4} \frac{E A_0}{L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$R = \int_0^L \rho w^2 A_0 \left(1 - \frac{x}{2L}\right) x \begin{Bmatrix} 1 - x/L \\ x/L \end{Bmatrix} dx$$

$$K_{11} U_1 + K_{12} U_2 = R_1$$

$$K_{21} U_1 + K_{22} U_2 = R_2$$

$$R_2 = \rho w^2 A_0 \left(\frac{L^3}{3L} - \frac{L^4}{8L^2} \right) = \frac{5 \rho w^2 A_0 L^2}{24}$$

$$\Rightarrow U_2 = \frac{5 \rho w^2 A_0 L^2}{24} \cdot \frac{4}{3} \frac{L}{EA_0} = \frac{5 \rho w^2 L^3}{18 E}$$

$$u = \left(1 - \frac{x}{L}\right) \times 0 + \frac{x}{L} \frac{5 \rho w^2 L^3}{18 E}$$

$$u = \frac{5 \rho w^2 L^2}{18 E} x, \text{ same as Ritz}$$