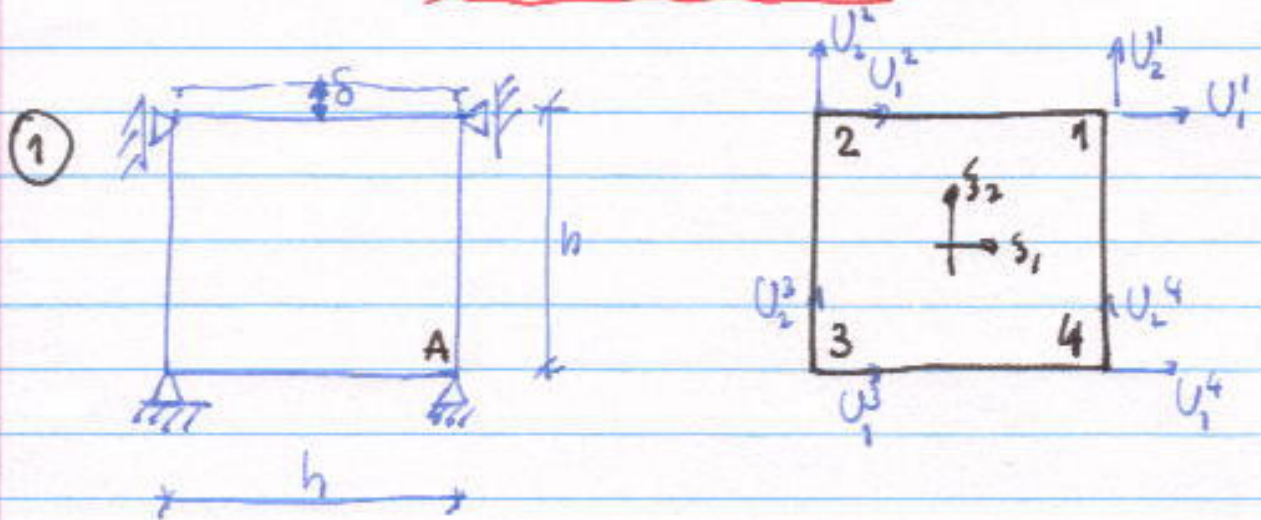


Practice Problems



$$[K^e] \{U^e\} = \{R^e\} \quad \{U^e\}^T = \{0 \ \delta \ 0 \ \delta \ 0 \ 0 \ 0 \ 0\}$$

The reaction at A (horizontal) is obtained from the equation:

$$K_{71} \times 0 + K_{72} \delta + 0 + K_{74} \delta + \dots = R_1^4 = H_A$$

$$\frac{E(1-\nu)}{8(2\nu^2+\nu-1)} \delta + \frac{E}{8(1+\nu)} \delta = R_1^4 = H_A$$

$$H_A = \frac{\nu E \delta}{2(1-\nu^2)}$$

Exact: $\epsilon_{22} = \frac{\delta}{h}$, $\epsilon_{11} = 0 = \frac{\sigma_{11}}{E} - \frac{\nu}{E}(\sigma_{22} + \sigma_{33})$
 $\epsilon_{33} = 0 = \frac{\sigma_{33}}{E} - \frac{\nu}{E}(\sigma_{11} + \sigma_{22})$

(2)

$$\begin{cases} \sigma_{11} = \nu (\sigma_{22} + \sigma_{33}) \\ \sigma_{33} = \nu (\sigma_{22} + \sigma_{11}) \end{cases}$$

$$\sigma_{11} + \sigma_{33} = 2\nu \sigma_{22} + \nu (\sigma_{11} + \sigma_{33}), \quad \sigma_{11} - \sigma_{33} = \nu (\sigma_{33} - \sigma_{11})$$

$$\sigma_{11} + \sigma_{33} = \frac{2\nu \sigma_{22}}{1-\nu} \quad \sigma_{11}(1+\nu) = \sigma_{33}(1+\nu)$$

$$E \epsilon_{22} = \frac{E \delta}{h} = \sigma_{22} - \nu \frac{2\nu \sigma_{22}}{1-\nu} = \sigma_{22} \frac{1-\nu-2\nu^2}{1-\nu}$$

$$\sigma_{22} = \frac{E \delta (1-\nu)}{h (1+\nu)(1-2\nu)}$$

$$2\sigma_{11} = \frac{2\nu}{(1-\nu)} \frac{E \delta (1-\nu)}{h (1+\nu)(1-2\nu)}$$

$$\sigma_{11} = \frac{\nu E \delta}{h (1+\nu)(1-2\nu)}$$

The reaction H_A is half the total load:

$$H_A = \frac{\nu E \delta b}{2} \frac{1}{h (1+\nu)(1-2\nu)}$$

$$H_A = \frac{\nu E \delta}{2} \frac{1}{(1+\nu)(1-2\nu)} \quad \text{SAME!!}$$

(2a)

$$\textcircled{2} \quad (EI w_0'')'' + (N w_0')' = 0 \quad 0 < x < L$$

Multiply by suitable virtual deflection field δw_0 and integrate between "0" and "L":

$$\int_0^L (EI w_0'')'' \delta w_0 dx + \int_0^L (N w_0')' \delta w_0 dx = \int_0^L \delta w dx$$

Integrate by parts:

$$\int_0^L \underbrace{[(EI w_0'')']}_{M} \delta w_0 dx - \int_0^L (EI w_0'')' \delta w_0' dx + \int_0^L (N w_0' \delta w_0)' dx$$

$$- \int_0^L N \delta w_0' w_0' dx = 0$$

$$\underbrace{(EI w_0'')'}_M \delta w_0 \Big|_0^L - \int_0^L (M \delta w_0')' dx + \int_0^L M \delta w_0'' dx +$$

$$+ N w_0' \delta w_0 \Big|_0^L - \int_0^L N w_0' \delta w_0' dx = 0$$

$$M' \delta w_0 \Big|_0^L + N w_0' \delta w_0 \Big|_0^L - M \delta w_0' \Big|_0^L + \int_0^L M \delta w_0'' dx -$$

$$- \int_0^L N w_0' \delta w_0' dx = 0$$

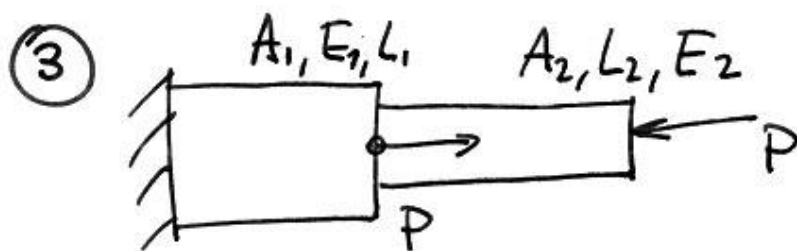
The PVD reads:

$$\int_0^L M \delta w_0'' dx = \int_0^L N w_0' \delta w_0' dx \quad 0 < x < L$$

$\forall \delta w_0$ s.t. $\delta w_0, \delta w_0'$ satisfy the
homogeneous essential
boundary conditions

The natural boundary conditions are not enforced by the principle.

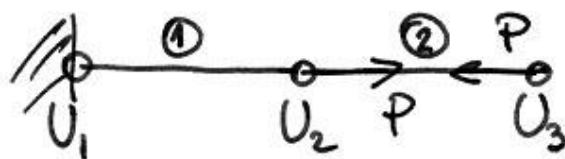
3a



By equilibrium reaction at built-in end is "0", as is the normal force in the left member.
The exact solution is:

$$\sigma_1 = 0 \quad \sigma_2 = \frac{P}{A_2}$$

Finite element solution:



$$K^e = \frac{A^e E^e}{L^e} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$\underbrace{\left[\begin{array}{c} \\ U_2 \\ U_3 \end{array} \right]}_K \begin{Bmatrix} 0 \\ U_2 \\ U_3 \end{Bmatrix} = \begin{Bmatrix} P \\ -P \end{Bmatrix}$$

$$\left\{ \begin{array}{l} K_{22} U_2 + K_{23} U_3 = P \\ K_{32} U_2 + K_{33} U_3 = -P \end{array} \right.$$

$$K_{22} = K_{22}^{(1)} + K_{11}^{(2)} = \frac{A_1 E_1}{L_1} + \frac{A_2 E_2}{L_2}$$

$$K_{23} = K_{12}^{(2)} = \frac{-A_2 E_2}{L_2} = K_{32}$$

$$K_{33} = K_{22}^{(2)} = \frac{A_2 E_2}{L_2}$$

Solving system:

$$(K_{22} + K_{32}) U_2 + 0 U_3 = 0 \Rightarrow \boxed{U_2 = 0}$$

$$K_{33} U_3 = -P$$

$$\rightarrow \boxed{U_3 = \frac{-PL_2}{A_2 E_2}}$$

Element ① $U_1^{(1)} = U_1 = 0, U_2^{(1)} = U_2 = 0$

$$u^{(1)} = \phi_1 \cdot 0 + \phi_2 \cdot 0 = 0$$

$$\varepsilon^{(1)} = 0, \sigma^{(1)} = 0$$

Element ② $U_1^{(2)} = U_2 = 0, U_2^{(2)} = U_3 = \frac{-PL_2}{E_2 A_2}$

$$u^{(2)} = \phi_1 \cdot 0 + \phi_2 \left(\frac{-PL_2}{E_2 A_2} \right), \phi_2 = \frac{x}{L_2}$$

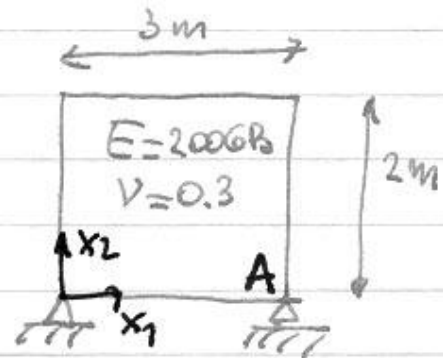
3c

$$\epsilon^{(2)} = \frac{-P}{E_2 A_2}, \quad \sigma^{(2)} = \frac{-P}{A_2} \quad \text{Exact!}$$

(4)

4 The 4-node plane strain element shown is subjected to the constant stresses

$$\begin{aligned}\sigma_{11} &= 200 \text{ MPa} \\ \sigma_{22} &= 100 \text{ MPa} \\ \sigma_{12} &= 100 \text{ MPa}\end{aligned} \quad (\text{loads not shown})$$



Compute the displacement at A

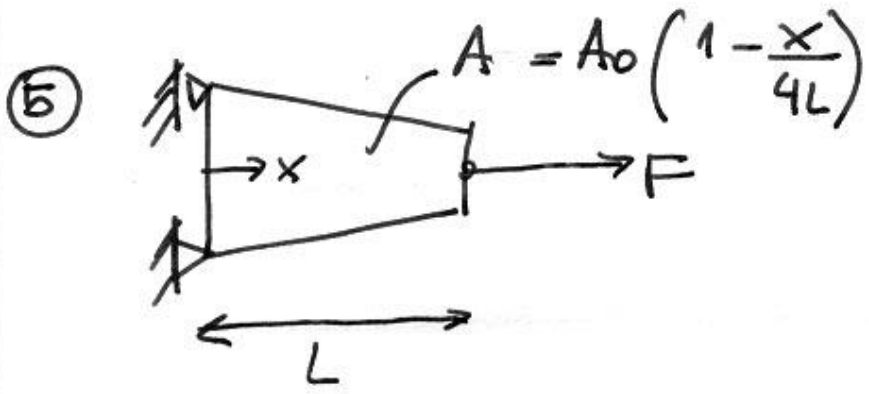
$$\epsilon_{22} = 0 \rightarrow \sigma_{33} = \nu(\sigma_{11} + \sigma_{22}) = 90 \text{ MPa}$$

$$\epsilon_{11} = \frac{1}{E} [\sigma_{11} - \nu(\sigma_{22} + \sigma_{33})] = \frac{143 \cdot 10^6}{200 \cdot 10^9}$$

$$u = x_1 \epsilon_{11} + x_2 2\epsilon_{12}$$

$$2\epsilon_{12} = \frac{\sigma_{12}}{G} = \frac{100 \text{ MPa}}{200 \text{ GPa}} \cdot 2(1 + 0.3) = \frac{2.6}{2 \cdot 10^3}$$

$$u(3, 0) = 3 \epsilon_{11} = \frac{3 \times 143}{200 \cdot 10^3} \text{ m} = 0.21 \text{ mm}$$



PVD: $\int_V \sigma_{ij} \delta \epsilon_{ij} dV = F \delta u(L)$

specializing to 1D: $\int_0^L A(x) \sigma \delta \epsilon dx = F \delta u(L)$

$\int_0^L A_0 \left(1 - \frac{x}{4L}\right) \left(\frac{72}{73} + \frac{24x}{73L}\right) \frac{F}{A_0} \delta \epsilon dx \stackrel{?}{=} F \delta u(L)$

① $\delta u = \frac{ax}{L}$ (admissible, satisfies homo. essential B.C. $\delta u(0) = 0$).

$\delta \epsilon = \frac{a}{L}$

PVD: $\int_0^L \left(1 - \frac{x}{4L}\right) \left(\frac{72}{73} + \frac{24x}{73L}\right) \frac{a}{L} dx \stackrel{?}{=} a$

$\frac{72}{73} - \frac{1}{4} \frac{72}{73} \frac{1}{2} + \frac{24}{73} \frac{1}{2} - \frac{24}{4 \times 73} \frac{1}{3} \stackrel{?}{=} 1$

$$\frac{7}{8} \frac{72}{73} + \frac{24}{73} \left(\frac{1}{2} - \frac{1}{12} \right) \stackrel{?}{=} 1$$

$$\left(\frac{21}{8} + \frac{5}{12} \right) \frac{24}{73} \stackrel{?}{=} 1$$

$$\frac{\cancel{63} + \cancel{10}}{\cancel{24}} \frac{\cancel{24}}{73} \stackrel{?}{=} 1 \quad \checkmark$$

$$\textcircled{2} \delta u = a x^3 / L^3, \quad \delta \epsilon = 3 a x^2 / L^3$$

$$\int_0^L \left(1 - \frac{x}{4L} \right) \left(\frac{72}{73} + \frac{24x}{73L} \right) \frac{3ax^2}{L^3} dx \stackrel{?}{=} \frac{1 \times 3}{3}$$

$$\frac{72}{73} \frac{1}{3} - \frac{1}{4} \frac{72}{73} \frac{1}{4} + \frac{24}{73} \frac{1}{4} - \frac{1}{4} \frac{24}{73} \frac{1}{5} \stackrel{?}{=} \frac{1}{3}$$

$$\frac{72}{73} \left(\frac{1}{3} - \frac{1}{16} \right) + \frac{24}{73} \left(\frac{1}{4} - \frac{1}{20} \right) \stackrel{?}{=} \frac{1}{3}$$

$$\frac{72}{73} \frac{13}{48} + \frac{24}{73} \frac{1}{5} \stackrel{?}{=} \frac{1}{3}$$

$$\frac{24}{73} \left(\frac{3 \times 13}{48 \cdot 16} + \frac{1}{5} \right) \stackrel{?}{=} \frac{1}{3}$$

$$\frac{24}{73} \left(\frac{65 + 16}{80} \right) = \frac{243}{730} = \boxed{0.3328 \neq \frac{1}{3}}$$

PVD not satisfied
stresses not in
equilibrium !!